

## Dynamic Analysis of Double Wishbone and Double Wishbone with S Link + Toe Link

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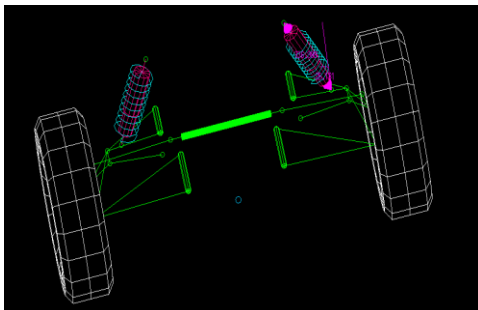
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### ABSTRACT:

Double wishbone or A arm are solid links used to connect the vehicles chassis to its wheels. These links provide better ride to the customers but in conditions of bumps and rebounds toe change increases drastically. This paper studies about the model of double wishbone used in off-road racing cars. Dynamic analysis of Wishbones is done in 'LOTUS-Shark V 5.01'. Analysis is also done by changing the design of double wishbone. By applying an extra link named toe link and S link for providing two mounting points.

**Keywords:** s-link, toe-link

### I. Introduction



Double wishbone is a type of suspension system which connects frame to tyres and also provides best handling Performance for the driver comfort we just introduce an extra link named "Toe Link" with that upper wishbone so toe changes minimum in the condition of bump steer, means when vehicle passes over a bump there is a condition of instability where toe changes instantly. By providing this toe link change in toe angle is nearly zero.

**S-Link** provides a connection between the upright and wishbone. It has two ports in frame direction and single towards tire side. With the help of ball joints, toe link and wishbone is connected to S-link on one side and a simple knuckle joint is wishbone. This link simply connects the upper wishbone to lower wishbone to restrict two plane motion of the upper wishbone. If it is not so then our vehicle can't be able to take turn. By providing this link, upper wishbone can move up and down only but toe link has two plane motion, provided between S-link and upright. Hence, there is only one plane motion between S-link and upright to steer our vehicle.

Another link is also provided between upper and lower

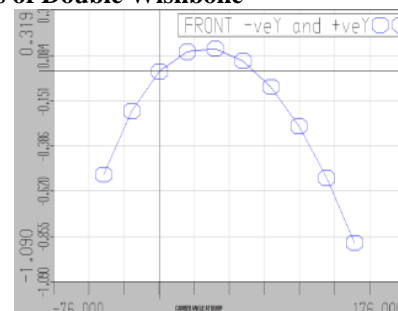
### II. Analysis of conventional or simple double wishbone

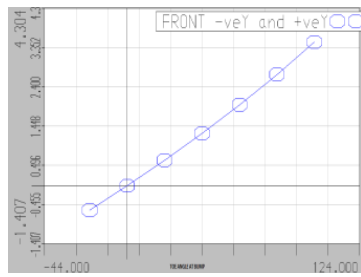
For the support of concept we have provided results of "LOTUS-SHARK V 5.01".

By taking same amount of the bump and rebound conditions results are plotted.

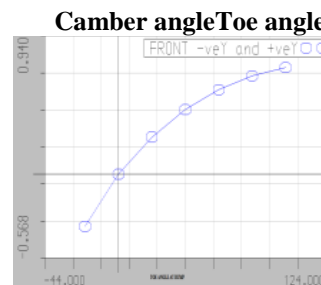
For conventional double wishbone toe change can be noticed from graphs and results.

#### Graphs of Double Wishbone



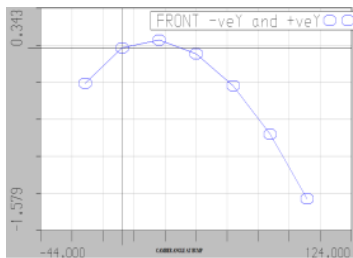


**Camber change Toe change**

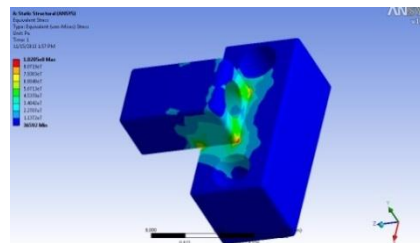
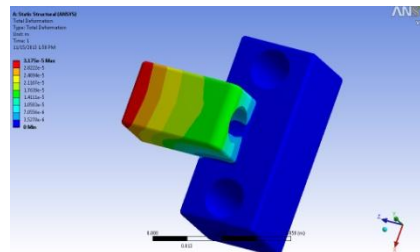


**Camber angle Toe angle**

Analysis of Double wishbone with toe link + S- Link

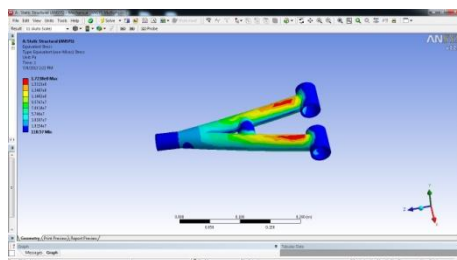
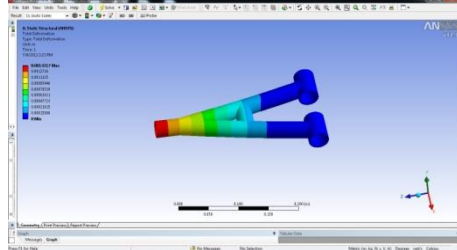


**Analysis of S-Link and upper wishbone**



Front Suspension  
Production.Shk

Filename: Final For Front



Deformation Stress concentration  
Fos :- 2.96

Deformation Stress concentration  
Fos:- 4.52

### III. Lotus results for wishbone

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### IV. Lotus Suspension Analysis V5.01

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TYPE 18 Double Wishbone, Upper toe link + S link

STATIC VALUES							
X	Y	Z					
(mm)	(mm)	(mm)					
3950.00	-180.00	280.00	POINT:1				Lower wishbone front pivot
4100.00	-180.00	280.00	POINT:2				Lower wishbone rear pivot
4000.00	-530.00	204.00	POINT:3				Lower wishbone outer ball joint
4000.00	-250.00	530.00	POINT:4				Upper wishbone front pivot
4100.00	-250.00	530.00	POINT:5				Upper wishbone rear pivot
4025.00	-450.00	480.00	POINT:6				Upper wishbone outer ball joint
4035.22	-395.05	478.06	POINT:7				Damper wishbone end
4096.70	-259.67	780.47	POINT:8				Damper body end

4130.00	-510.00	300.00	POINT:9	Outer	4030.00	440.00	190.00	POINT:216	Part 1
track rod ball joint					C of G(2)				
4100.00	-210.00	360.00	POINT:10	Inner	4170.00	480.00	450.00	POINT:217	Part 2
track rod ball joint					C of G(2)				
4000.00	-550.00	292.00	POINT:13	Wheel	4230.00	525.00	220.00	POINT:218	Part 3
spindle point					C of G(2)				
4000.00	-600.00	292.00	POINT:14	Wheel	4070.00	630.00	440.00	POINT:219	Part 4
centre point					C of G(2)				
3900.00	-260.00	550.00	POINT:15	Upper	4020.00	510.00	455.00	POINT:220	Part 5
toe link inboard end					C of G(2)				
3975.00	-450.00	480.00	POINT:16	Upper	4130.00	725.00	260.00	POINT:221	Part 6
toe link outboard end					C of G(2)				
4000.00	-530.00	380.00	POINT:17	Drop					
link axis point									
4030.00	-440.00	190.00	POINT:18	Part 1					
C of G									
4170.00	-480.00	450.00	POINT:19	Part 2					
C of G									
4230.00	-525.00	220.00	POINT:20	Part 3					
C of G									
4070.00	-630.00	440.00	POINT:21	Part 4					
C of G									
4020.00	-510.00	455.00	POINT:22	Part 5					
C of G									
4130.00	-725.00	260.00	POINT:23	Part 6					
C of G									
3950.00	180.00	280.00	POINT:201						
Lower wishbone front pivot(2)									
4100.00	180.00	280.00	POINT:202						
Lower wishbone rear pivot(2)									
4000.00	530.00	204.00	POINT:203						
Lower wishbone outer ball joint(2)									
4000.00	250.00	530.00	POINT:204						
Upper wishbone front pivot(2)									
4100.00	250.00	530.00	POINT:205						
Upper wishbone rear pivot(2)									
4025.00	450.00	480.00	POINT:206						
Upper wishbone outer ball joint(2)									
4035.22	395.05	478.06	POINT:207						
Damper wishbone end(2)									
4096.70	259.67	780.47	POINT:208						
Damper body end(2)									
4130.00	510.00	300.00	POINT:209	Outer					
track rod ball joint(2)									
4100.00	210.00	360.00	POINT:210	Inner					
track rod ball joint(2)									
4000.00	550.00	292.00	POINT:211						
Wheel spindle point(2)									
4000.00	600.00	292.00	POINT:212						
Wheel centre point(2)									
3900.00	260.00	550.00	POINT:213						
Upper toe link inboard end(2)									
3975.00	450.00	480.00	POINT:214						
Upper toe link outboard end(2)									
4000.00	530.00	380.00	POINT:215	Drop					
link axis point(2)									

STATIC VALUES

Camber Angle (deg):	0.00
Toe Angle {Plane} (deg):	0.00
Toe Angle {SAE} (deg):	0.00
Castor Angle (deg):	0.00
Castor Trail (hub) (mm):	0.00
Castor Offset (grnd) (mm):	0.00
Kingpin Angle (deg):	0.00
Kingpin Offset (w/c) (mm):	70.00
Kingpin Offset (grnd) (mm):	70.00
Mechanical Trail (grnd) (mm):	0.00
ROLL CENTRE HEIGHT (mm):	112.96

GENERAL DATA VALUES

TYRE ROLLING RADIUS (mm):	292.00
WHEELBASE (mm):	1550.00
C OF G HEIGHT (mm):	510.00
BREAKING ON FRONT AXLE (%) :	60.00
DRIVE ON FRONT AXLE (%) :	0.00
WEIGHT ON FRONT AXLE (%) :	40.00
OUTBOARD FRONT BRAKES:	
OUTBOARD REAR BRAKES:	
INDEPENDENT FRONT SUSPENSION:	
INDEPENDENT REAR SUSPENSION:	
RACK TYPE STEERING ARTICULATION:	

RUN DETAILS

FULL MODEL:			
BUMP TRAVEL (mm):	100.00		
INCREMENT (mm):	20.00		
REBOUND TRAVEL (mm):	20.00		
INCREMENT (mm):	20.00		
ROLL ANGLE (deg):	3.00	ROLL	
INCREMENT (deg):	0.50		
STEERING TRAVEL (mm):	50.00		
STEERING INCREMENT (mm):	5.00		

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LOTUS SUSPENSION ANALYSIS v5.01
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FRONT SUSPENSION - BUMP TRAVEL
LHS WHEEL (-ve Y)

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60.00	-0.22	0.00	125.16	65.16	8.94
-0.97	-48.29	-48.29			
40.00	-0.29	0.00	122.23	82.23	6.51
-0.73	-32.26	-32.26			
20.00	-0.39	0.00	118.50	98.50	3.52
-0.42	-16.20	-16.20			
0.00	-0.51	0.00	112.96	112.96	0.00
0.00	0.00	0.00			
-20.00	-0.71	0.00	103.72	123.72	-3.97
0.60	16.47	16.47			

TYPE 18 Double Wishbone, Upper toe link + S link

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INCREMENTAL GEOMETRY VALUES

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LOTUS SUSPENSION ANALYSIS v5.01

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Bump Damper1 Travel Ratio (mm) (-)	Camber Spring1 Angle (deg) (deg)	Toe {SAE}	Castor (deg)	Kingpin Angle (deg)
100.00	-1.3045	0.7244	0.2104	1.3072
1.230 1.230				
80.00	-0.7469	0.6674	0.1938	0.7492
1.240 1.240				
60.00	-0.3263	0.5738	0.1666	0.3280
1.246 1.246				
40.00	-0.0497	0.4388	0.1273	0.0506
1.248 1.248				
20.00	0.0683	0.2532	0.0734	-0.0680
1.242 1.242				
0.00	0.0000	0.0000	0.0000	0.0000
1.225 1.225				
-20.00	-0.3048	-0.3530	-0.1022	0.3054
1.205 1.205				

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FRONT SUSPENSION - BUMP TRAVEL

RHS WHEEL (+ve Y)

TYPE 18 Double Wishbone, Upper toe link + S link

INCREMENTAL GEOMETRY VALUES

Bump Damper1 Travel Ratio (mm) (-)	Camber Spring1 Angle (deg) (deg)	Toe {SAE}	Castor (deg)	Kingpin Angle (deg)
100.00	-1.3045	0.7244	0.2104	1.3072
1.230 1.230				
80.00	-0.7469	0.6674	0.1938	0.7492
1.240 1.240				
60.00	-0.3263	0.5738	0.1666	0.3280
1.246 1.246				
40.00	-0.0497	0.4388	0.1273	0.0506
1.248 1.248				
20.00	0.0683	0.2532	0.0734	-0.0680
1.242 1.242				
0.00	0.0000	0.0000	0.0000	0.0000
1.225 1.225				
-20.00	-0.3048	-0.3530	-0.1022	0.3054
1.205 1.205				

**V. Incremental Suspension Parameter Values**

Bump Wheelbase Travel Change (mm)	Anti Damper1 Dive Travel (mm)	Anti Spring1 Squat Travel (mm)	Roll Centre (mm)	Roll Centre (mm)	Half Track (mm)
100.00	-0.08	0.00	130.86	30.86	12.07
-1.29	-80.57	-80.57			
80.00	-0.15	0.00	127.89	47.89	10.79
-1.16	-64.38	-64.38			

100.00	-1.3045	0.7244	0.2104	1.3072
1.230 1.230				
80.00	-0.7469	0.6674	0.1938	0.7492
1.240 1.240				
60.00	-0.3263	0.5738	0.1666	0.3280
1.246 1.246				
40.00	-0.0497	0.4388	0.1273	0.0506
1.248 1.248				
20.00	0.0683	0.2532	0.0734	-0.0680
1.242 1.242				
0.00	0.0000	0.0000	0.0000	0.0000
1.225 1.225				
-20.00	-0.3048	-0.3530	-0.1022	0.3054
1.205 1.205				

INCREMENTAL SUSPENSION PARAMETER VALUES

						40.00	-0.29	0.00	122.23	82.23	6.51
Bump	Anti	Anti	Roll	Roll	Half	-0.73	-32.26	-32.26			
Wheelbase	Damper1	Spring1				20.00	-0.39	0.00	118.50	98.50	3.52
Travel	Dive	Squat	Centre	Centre	Track	-0.42	-16.20	-16.20			
Change	Travel	Travel				0.00	-0.51	0.00	112.96	112.96	0.00
(mm)	(%)	(%)	Height {to	Height {to		0.00	0.00	0.00			
Change	(mm)	(mm)	(mm)Grnd}	(mm)	(mm)	-20.00	-0.71	0.00	103.72	123.72	-3.97
		Body}				0.60	16.47	16.47			
100.00	-0.08	0.00	130.86	30.86	12.07	*****					
-1.29	-80.57	-80.57				*****					
80.00	-0.15	0.00	127.89	47.89	10.79						
-1.16	-64.38	-64.38									
60.00	-0.22	0.00	125.16	65.16	8.94						
-0.97	-48.29	-48.29									

### VI. Conclusion

It is clear from the graphs there is a small or negligible change in the toe angle when vehicle faces the condition of bump and rebound. Moreover, the height of the rack mounting is also low with double wishbone +slink +toe link. Therefore in condition of simple Ackermann if rack is placed behind the pedals (especially in off-road vehicles) Comfort ability increases to large extend for driver because pedals can be operated without facing any obstacles.

### References

- [1] LOTUS SHARK V-5.01
- [2] Anonymous (1992) Directive 92/7/EC, (on definition and testing of 'road friendly suspension'), Council of the European communities.
- [3] Bastow D.(1980) car suspension and handling, 1<sup>st</sup>edn., Pentech press